

**HIGHLY COMPACT, PRECISION LIGHTWEIGHT DEPLOYABLE  
TRUSS WHICH ACCOMMODATES SIDE MOUNTED COMPONENTS**

**FIELD OF THE INVENTION**

[001] The present invention relates in general to space-deployable structures, and is particularly directed to a lightweight truss structure which accommodates the side mounting of components in its deployed configuration, and which folds to a highly nested, compact stacked configuration when stowed.

**BACKGROUND OF THE INVENTION**

[002] In order to transport and space-deploy large physical structures, such as antennas, solar reflectors and the like, using cost effective (small) launch vehicles, it is necessary that the underlying support architecture for the deployed structure be lightweight and compactly stowable in as small a payload volume as possible. Many of the space deployment architectures that have been proposed to date employ a relatively long (on the order of three hundred meters or more) rectilinear boom, that provides for the mounting of a variety of devices along its length. Moreover, many

applications which use a boom require the boom to be extremely lightweight and have a high degree of stiffness or rigidity. This is particularly true in the case of large antennas, which need to be precisely deployed and must maintain geometry precision on orbit. For such applications it is also necessary that the deployment of the boom be rate and geometry controlled.

**[003]** Unfortunately, the relatively large, high stiffness booms that have been proposed and deployed to date typically use canister mechanisms for their deployment that are relatively heavy and do not allow side mounting of payloads along the entire length of the structure. Telescoping booms are an alternative, yet like canister deployed structures, they have no side mounting capability. Inflatable structures, on the other hand, provide for highly compact stowage; however, once deployed they are subject to micro-meteoroid damage; they also lack geometric precision due to the fact that they have a relatively high coefficient of thermal expansion (CTE). To address the deployed geometry precision problem, rigidized inflatables have been suggested. However, these structures suffer from fiber breakage, a lack of deployment repeatability and final material characteristic consistency.

#### **SUMMARY OF THE INVENTION**

**[004]** In accordance with the present invention, shortcomings of conventional space-deployable boom structures, such as those described above, are

effectively obviated by means of a collapsible truss structure, that is rectilinearly deployable from a tightly nested, stowed configuration to an elongated truss configuration. As will be described, the truss structure of the present invention contains a plurality of foldable, truss-forming multi-sided bays. Each bay contains a pair of multi-sided (e.g., triangular) battens that are joined together at corner regions thereof by foldable longerons.

**[005]** In addition, each side of a bay contains a plurality of flexible cord diagonal members that cross one another and connected to diagonally opposed corner regions of that side. When the longerons are in their folded positions, the battens are nested together against one another in a stacked arrangement and the flexible cord diagonal members flex into a compact stowed configuration between adjacent battens.

**[006]** Each corner region of a batten includes a pair of flexible clamps that are configured to engage an elongated support member in the stowed configuration of the bay containing that batten. In the course of deployment of the bay outwardly from its stowed configuration, the clamps travel along and leave the elongated support member, and engage threads of an elevator screw that is coaxial with and extends outwardly from said elongated support member.

**[007]** The elevator screw is coaxial with an elongated lead screw, which passes through said elongated support member, such that rotation of said elongated lead screw

initially causes linear travel of the elevator screw over a prescribed distance, sufficient to deploy the outermost bay of the truss. The elevator screw then becomes fixedly engaged with or slaved to the lead screw. Once this occurs, further rotation of the lead screw causes rotation of the elevator screw therewith. The clamps travel along the elevator screw until they leave the elevator screw in the course of deployment of a respective bay of the truss structure. Just prior to a batten frame leaving the elevator screw the next batten of the folded truss structure is pulled onto and engaged with the elevator screw.

**[008]** Rotation of the lead screws are controlled by a single drive motor. The output shaft of the drive motor is coupled to a gearing and interconnect arrangement that is coupled to each of the lead screws and is retained by a baseplate from which the elongated tubular support members extend. Operation of the motor drives the gearing and interconnect arrangement, so as to cause synchronized rotation of each of the lead screws and the elevator screws engaged thereby, thereby sequentially deploying successively adjacent bays of the truss structure.

**[009]** Prior to deployment of the truss structure the folded assembly is stored by a compressive load from a tensioned cable seating each batten to the adjacent battens at the cup cones. This load allows the stowed truss system to tolerate and transfer inertial loads generated by its own mass and those of payloads attached

to each bay to its mounting point at its base. This capability allows the deployment device to be sized for only its deployment functions and not to tolerate the loads of the truss under dynamic loads.

**BRIEF DESCRIPTION OF THE DRAWINGS**

[010] Figure 1 is a diagrammatic perspective view of the deployed configuration of an individual bay of the truss of the present invention;

[011] Figure 2 is partial perspective view of the deployed configuration of multiple bays of the rectilinear truss structure of the invention;

[012] Figure 3 is a perspective view of a partially deployed configuration of the rectilinear truss structure of the invention;

[013] Figure 4 is a diagrammatic front view of a batten in its collapsed or stowed condition in the truss structure of the invention;

[014] Figure 5 is an enlarged partial perspective view of the distal portion of the corner region of a respective batten of the truss structure of the invention;

[015] Figure 6 is a partial side view of the stowed configuration of the truss structure of the invention showing cup cone assemblies that provide separation and load transfer prior to deployment between sequentially adjacent battens;

[016] Figure 7 diagrammatically illustrates a cone-cup shape for the stand-offs of Figure 6;

[017] Figures 8 and 9 are diagrammatic perspective views of a corner fitting installable at a corner region of a batten of the truss structure of the invention;

[018] Figure 10 is a partial side view diagrammatically illustrating the configuration of an elevator and lead screw arrangement as retained within a stowage tube of the truss structure of the invention;

[019] Figure 11 is a diagrammatic perspective view of the coupling of a drive motor to respective lead screws at corner locations of a base plate of the truss structure of the invention;

[020] Figure 12 is an enlarged partial end view of the gear arrangement coupling of the output shaft of the drive motor of Figure 11 to lead screw-driving torque tubes;

[021] Figures 13-17 diagrammatically illustrate the sequential manner in which the truss structure of the invention is deployed from its stowed configuration; and

[022] Figures 18 and 19 are respective perspective views of a pair of battens in their collapsed and partially deployed states, respectively.

#### **DETAILED DESCRIPTION**

[023] Attention is initially directed to Figure 1, which is a diagrammatic perspective view of the deployed configuration of an individual bay of the truss of the present invention. As described briefly above, and as further shown in the partial perspective view of Figure 2, the rectilinear truss structure of the invention is

comprised of a plurality of such bays that are sequentially interconnected with one another by means of sets of hinged longerons, which are foldable between successive battens of the truss. More particularly, as shown in Figure 1, the ends of a respective truss bay are defined by a pair of multi-sided, rigid frames or battens 10 and 11. In accordance with a non-limiting, but preferred embodiment, each batten is preferably formed as a laminate of layers of graphite composite material and has a generally triangular configuration. It should be observed, however, that other materials and geometries may be employed without departing from the invention. The use of a triangular configuration is a preferred geometry as it serves to limit the overall size and therefore payload weight and complexity of the bay, while providing the intended truss structure and ability to side mount components.

**[024]** Triangular batten 10 is formed of three sides F1, F2 and F3, while triangular batten 11 is formed of three sides F4, F5 and F6. In accordance with a preferred embodiment, each of the sides of a respective batten has the same length, so that the geometry of a respective batten is essentially that of an equilateral triangle. Battens 10 and 11 are connected with one another by three parallel and foldable/hinged tubular or hollow rod-shaped longerons L1, L2 and L3, that connect like corners regions of the battens with one another. In particular, longeron L1 connects corner C13 formed at the intersection of sides F1 and F3 of batten 10 with

corner C46 formed at the intersection of sides F4 and F6 of batten 11. Longerons L2 connects corner C12 formed at the intersection of sides F1 and F2 of batten 10 with corner C45 formed at the intersection of sides F4 and F5 of batten 11. Likewise, longeron L3 connects corner C23 formed at the intersection of sides F2 and F3 of batten 10 with corner C56 formed at the intersection of sides F5 and F6 of batten 11. Like battens 10 and 11, the longerons are preferably made of graphite composite material. In addition, the longerons are hinged at their midpoints to facilitate stowage and deployment as will be described.

**[025]** Also shown in Figure 1 are three pairs of flexible diagonal rods or cords, which interconnect diagonally opposing corners of the battens. Like the battens and the longerons, the diagonals are preferably made of graphite composite material. As shown in the perspective view of Figure 3 and the diagrammatic front view of Figure 4, in the collapsed or stowed condition of the truss, the hinged longerons are effectively folded 'in-half', and the diagonal cords relax between the sides of the battens; in the deployed condition of the truss (Figures 1 and 2), the longerons unfold to their full lengths and the diagonals are placed in tension and are generally located within the confines of respective rectangles defined by opposing pairs of batten sides and longerons therebetween.

**[026]** In particular, a diagonal D1 connects corner C13 of batten 10 with diagonally opposite corner C45 of batten



11; while diagonal D2, which crosses diagonal D1, connects corner C12 of batten 10 with corner C46 of batten 11. Similarly, diagonal D3 connects corner C23 of batten 10 with diagonally opposite corner C46 of batten 11; and diagonal D4, which crosses diagonal D3, connects corner C13 of batten 10 with corner C56 of batten 11. Likewise, diagonal D5 connects corner C23 of batten 10 with diagonally opposite corner C45 of batten 11; and diagonal D6, which crosses diagonal D5, connects corner C12 of batten 10 with corner C56 of batten 11.

**[027]** As described earlier, and as shown generally at 21-26 in Figure 1 and in enlarged detail in the partial perspective view of Figure 5, the distal portion of the corner region of a respective batten contains a pair of mutually opposing, generally C-shaped, flexible clamps 30 and 40. These clamps are sized to flexibly engage and be slidable along the outer surface of a generally round structural tube 50 in the stowed configuration of the truss, and to engage threads of an elevator screw 60, which extends axially outwardly from the stowage tube in the course of deployment of the truss. For this purpose, the C-clamps 30, 40 are provided with sets of thread slots 32 and 42, respectively, that are sized and shaped to conform with and engage the threads of the elevator screw 60.

**[028]** Disposed adjacent to the C-clamps are respective tubular shaped stand-offs 35 and 45. As shown in the partial side view of Figure 6, these stand-offs are sized to provide a prescribed separation 55 between

sequentially adjacent battens in the stowed configuration of the truss. As further shown in Figure 7, in order to facilitate mutual engagement therebetween, one of the mutually facing pair of stand-offs (cup cone) may have a generally cone configuration, while the other stand-off may have a generally cup configuration complementary to the cone configuration of its opposing stand-off.

**[029]** In order to connect the hinged longerons and the flexible diagonals to the battens, a respective corner region of a batten has a generally elongated slot, shown at 37 in Figure 5. This slot is sized to receive a corner fitting 70, depicted in perspective in Figure 8. As shown therein, a respective corner fitting 70 has a clevis 71 that is sized to fit and be captured within the slot 37, by means of screws and the like. The clevis includes a pair of opposite slots 72 and 73, that are sized to receive longeron end-fittings 80, one of which is shown in Figure 8. Bores 82 and 83 are formed in the clevis 71 and are sized to receive pins that pass through corresponding bores (not shown) in shaft portions 85 of the longeron end-fittings, so as to allow the longerons to pivot about the axes of the bores, as shown as Figure 9. The shaft portion 85 of a respective longeron end-fitting terminates at a disc portion 87 of the longeron end-fitting. The disc portion 87 of a longeron end-fitting has a generally circular mesa portion 88, that is sized to fit within and be bonded to

the open end of a longeron, thereby pivotally capturing an end of a longeron at a corner region of a batten.

**[030]** As further shown in Figure 8, a respective corner fitting further includes a ball seat element 90, having a central aperture 91 that receives a boss 75 of the corner fitting 70. The ball seat element 90 includes a set of four corner apertures 92-95 that are sized to receive associated ball-shaped fittings 100 terminating respective ends of the diagonal cords. A ball seat element 90 further includes a set of four diagonal cord guide slots 102-105 that extend between the outer surface of the ball seat element and the corner apertures 92-95 thereof. The diagonal cord guide slots 102-105 serve to allow for the proper orientation of the distal ends of the diagonal cords for the stowed and deployed configurations of the battens. A fastener 109, such as a screw or the like is used to secure the ball seat element 90 to the corner fitting 70.

**[031]** As pointed out briefly above, deployment of a respective batten is accomplished by means of an elevator screw that becomes engaged by the pairs of C-clamps at the distal ends of the corner regions of the batten. As shown in Figure 10, the elevator screw 60 is retained within and is coaxial with structural tube 50. An interior end 61 of the elevator screw is terminated by a nut 62 having a threaded bore 63 that is coaxial with the elevator screw 60. A lead screw 110, in the form of a hollow rod with a threaded exterior surface, engages the threads of the nut 62 of the elevator screw,

such that rotation of the lead screw 110 may cause rectilinear travel of the elevator screw 60 along the interior of the structural tube 50.

**[032]** The nut 62 has a radial bore 64 that contains a spring-loaded pin 65. This pin is sized to engage an associated detent in the lead screw 110, when the elevator screw has been translated to its outermost extension position from the structural tube 50, making the elevator screw solid with, or slaved to, the lead screw at this point in the travel of the elevator screw. This outermost extension position of the elevator screw 60 is slightly longer than the length of a respective truss bay, so that a bay may acquire its deployed configuration as its two end battens engage the elevator screw. Once the elevator screw 60 becomes slaved to the lead screw 110, rotation of the elevator screw 60 will cause an associated rotation of the elevator screw. This, in turn, will cause outward translation of a batten, whose C-clamps engage the elevator screw.

**[033]** As shown in Figure 11, rotation of the lead screw 110 is accomplished by means of a motor 120, which is mounted to a corner region 131 of a base plate 130. As further shown in enlarged detail in the partial end view of the motor mount in Figure 12, the output shaft 121 of motor 120 is coupled to a gear arrangement 140 which, in turn is coupled to a pair of drive shafts (torque tubes) 141 and 142, which are terminated at distal ends thereof by means of gearing arrangements 150 and 160. The gear arrangements 140, 150 and 160 have respective output

shafts 145, 155 and 165 that serve as lead screws described above.

**[034]** The manner in which the truss structure of the invention is deployed from its stowed configuration is diagrammatically illustrated in Figures 13-17. Figure 13 shows the truss structure in its stowed or fully retracted configuration, wherein the elevator screw 60 projects slightly beyond the outer end of the structural tube 50 and is engaged by the C-clamps 30, 40 of a first or outermost batten B1. The diagrammatic perspective view of Figure 18 shows the manner in which a pair of battens B1 and B2 and the interconnecting longerons and diagonals thereof are collapsed in a juxtaposed manner. In this stowed configuration, the C-clamps of the remaining battens engage the outer surface of the structural tube 50. To begin sequential deployment of the bays of the truss, drive motor 120 is energized.

**[035]** Operation of the drive motor 120 causes its drive shaft and associated gear arrangements 140, 150 and 160 described above to rotate the drive shafts/lead screws 145, 155 and 165. As the lead screws are rotated by the operation of the motor 120, their associated elevator screws 60 are translated axially outwardly away from the stowed set of battens, thereby translating the outermost batten B1 away from the stowed stack, causing partial deployment of the first truss bay, as shown in Figure 14, and in the diagrammatic perspective view of Figure 19 for the pair of battens B1 and B2.

**[036]** Eventually, as shown in Figure 15, the outermost batten B1 becomes translated sufficiently to cause complete deployment of the first bay to the condition shown in Figure 1, described above, with the C-clamps of the outermost batten B1 being positioned adjacent to the distal ends of the elevator screws 60, and the C-clamps of the next batten B2 still being retained on the structural tube 50. At this point the elevator screws 60 become solid with the lead screws, so that further rotation of the lead screws will cause rotation, rather than translation, of the elevator screws.

**[037]** Next, as shown in Figure 16, as the elevator screws are further rotated by the rotation of the lead screws to which they are slaved, they translate the first bay closer to the outermost ends of the elevator screws. This translation of the first bay and thereby the second batten B2 thereof (which serves as the outermost batten of the second bay) serves to deploy the second truss bay, as the second batten B2 is translated off the structural tube 50. The C-clamps of the second batten B2 now engage the threads of the rotating elevator screws 60. Next, as shown in Figure 17, further rotation of the lead screws and elevator screws slaved thereto cause the outermost batten B1 to axially depart from the distal ends of the elevator screws, as the second batten B2 is translated along the elevator screws, partially deploying the second bay of the truss.

**[038]** With further rotation of the elevator screws, the second bay becomes fully deployed, and the third bay

will begin to deploy. Next, the batten B2 that interconnects the first and second bays will axially depart from the distal ends of the elevator screws, in the same manner as the outermost batten B1, as described above, and the above sequence of events will continue until all of the bays have been fully deployed.

**[039]** While we have shown and described an embodiment in accordance with the present invention, it is to be understood that the same is not limited thereto but is susceptible to numerous changes and modifications as known to a person skilled in the art, and we therefore do not wish to be limited to the details shown and described herein, but intend to cover all such changes and modifications as are obvious to one of ordinary skill in the art.